

10/679,629

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9-1-06

AMENDMENTS TO THE SPECIFICATION

On Page 1, ~~at Lines 4 and 5, between "body." and~~
~~"BACKGROUND"~~, please insert the following title and paragraph:
with

--CROSS REFERENCE TO RELATED APPLICATION

--This application is a divisional application of United
States Patent Application Serial Number 10/032,652 filed
November 13, 2001.--

On Page 7, please replace Lines ¹⁷~~10~~ to 19, with:

--Figure 4 depicts a perspective view of the panel
~~fastening device 120~~ panel edge joining device 140 for
panels 142 of cover 100 of this invention.--

On Page ¹⁴~~8~~, please replace Lines ⁷~~8~~ to 16, with:

-Also as a part of ground holding device 120 is a panel
edge joining ~~system device~~ device 140, for those situations when the
turf cover 100 is not one piece. Panel edge joining
device 140 permits one panel 142 to be joined to at least one
other panel 142 in order to form winter turf cover 100, and
maintain the required impermeability. Panel edge joining
device 140 may be situated on one, two, three, or four sides
of the preferably rectangular fabric 122. With panel edge
joining device 140, sheets of rectangular fabric 122 may be
joined to form winter turf cover 100.--

spec

Appl. No.: 10/716,651
Amdt. dated 08/11/2006
Reply to Office action of 06/30/2006

According to another embodiment of the present invention, there is provided a method of manufacturing a rotor structured to rotate with a flow of gas through a housing. The method includes providing first parameters defining a geometric configuration of a blade extending radially from the rotor and defining an edge, providing second parameters defining an expected cyclic pressure distribution on the blade during rotation of the rotor in the housing, determining a high displacement portion of the blade being subjected to a relatively higher displacement than adjacent portions of the blade resulting from the expected cyclic pressure distribution, adjusting the first parameters to remove at least part of the high displacement portion from the blade such that the edge of the blade is nonlinear in radial-axial projection, and thereafter forming the blade according to the first parameters.

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9-20
Please replace the paragraph on page 11, lines 5-16; with the following replacement paragraph:

The adjustment of the profile of the second edge 38 need not conform precisely to the portion 46a of the blade 32a that is subjected to relatively high displacements. Instead, the adjustment of the profile can also be determined in consideration of the strength of the blade 32, the ease of casting or otherwise forming the blade 32, the aerodynamic performance of the blade 32 and, hence, the rotor 30, and additional considerations. For example, as shown in Figures 1 and 3, the profile in the generally axial direction can define a smooth and continuous curve from a first end to a second end in order to minimize sharp edges that might otherwise concentrate stress and/or induce unnecessary pressure losses. The change in the profile of the edge 38 can also result in a reduction in the vibrating mass of the rotor 30, which typically increases the natural vibratory frequencies of the rotor 30, possibly increasing one or more of the resonant frequencies of the rotor 30 beyond the operating frequency of the rotor 30.